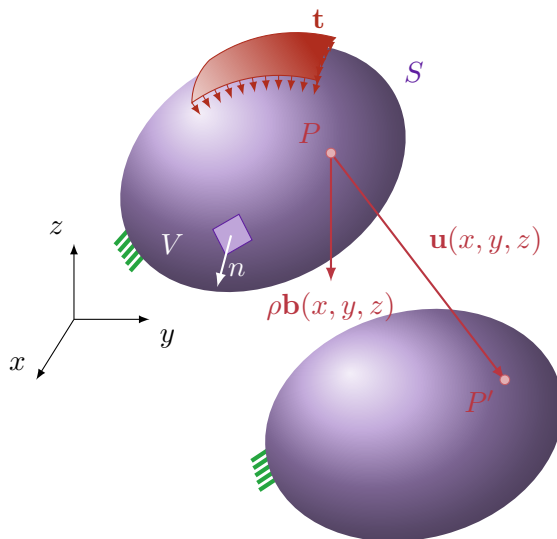


2. MES w mechanice ośrodka ciągłego

2.1. Stan równowagi



Wektor gęstości sił masowych $[\text{N}/\text{m}^3]$

$$\rho \mathbf{b} = \rho \begin{bmatrix} 0 \\ 0 \\ -g \end{bmatrix}$$

Wektor gęstości sił powierzchniowych $[\text{N}/\text{m}^2]$

$$\mathbf{t} = \begin{bmatrix} t_x \\ t_y \\ t_z \end{bmatrix}$$

Równanie równowagi ciała

$$\int_S \mathbf{t} dS + \int_V \rho \mathbf{b} dV = 0$$

Statyczne warunki brzegowe

$$\mathbf{t} = \boldsymbol{\sigma} \mathbf{n}$$

gdzie $\boldsymbol{\sigma}$ – tensor naprężeń

Wykorzystując twierdzenie Greena–Gaussa–Ostrogradzkiego

$$\int_S \boldsymbol{\sigma} \mathbf{n} dS = \int_V \text{div} \boldsymbol{\sigma} dV$$

Równania Naviera

$$\int_V (\operatorname{div} \boldsymbol{\sigma} + \rho \mathbf{b}) dV = 0 \iff \operatorname{div} \boldsymbol{\sigma} + \rho \mathbf{b} = 0 \quad \forall P \in V$$

$$\sigma_{ij,j} + \rho b_i = 0 + \text{w.b.}$$

Notacja tensorowa

$$t = \boldsymbol{\sigma} \mathbf{n} \quad \text{gdzie } \boldsymbol{\sigma} = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{xy} & \sigma_y & \tau_{yz} \\ \tau_{xz} & \tau_{yz} & \sigma_z \end{bmatrix}, \quad \mathbf{n} = \begin{bmatrix} n_x \\ n_y \\ n_z \end{bmatrix}$$

Notacja Voigt'a

$$t = \mathbf{m}^T \mathbf{s} \quad \text{gdzie } \mathbf{s} = \begin{bmatrix} \sigma_x \\ \sigma_y \\ \sigma_z \\ \tau_{yz} \\ \tau_{xz} \\ \tau_{xy} \end{bmatrix}, \quad \mathbf{m} = \begin{bmatrix} n_x & 0 & 0 \\ 0 & n_y & 0 \\ 0 & 0 & n_z \\ 0 & n_z & n_y \\ n_z & 0 & n_x \\ n_y & n_x & 0 \end{bmatrix}$$

$$\operatorname{div} \boldsymbol{\sigma} = \mathbf{L}^T \mathbf{s} \quad \text{gdzie } \mathbf{L} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 & 0 \\ 0 & \frac{\partial}{\partial y} & 0 \\ 0 & 0 & \frac{\partial}{\partial z} \\ 0 & \frac{\partial}{\partial z} & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial z} & 0 & \frac{\partial}{\partial x} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} & 0 \end{bmatrix} \quad \text{gdzie } \mathbf{L} - \text{macierz operatorów różniczkowych}$$

Równanie Naviera w notacji Voigt'a

$$\mathbf{L}^T \mathbf{s} + \rho \mathbf{b} = 0$$

Sformułowanie słabe – funkcja wagowa $w \cong \delta \mathbf{u}$ – kinematycznie dopuszczalna wariacja przemieszczenia (zgodna z kinematycznymi warunkami brzegowymi – **zasada prac wirtualnych**)

$$\int_V (\delta \mathbf{u})^T (\mathbf{L}^T \mathbf{s} + \rho \mathbf{b}) dV = 0 \quad \forall \delta \mathbf{u}$$

$$- \int_V (\mathbf{L} \delta \mathbf{u})^T \mathbf{s} dV + \int_S (\delta \mathbf{u})^T \overset{\mathbf{t}}{\mathbf{m}^T \mathbf{s}} dS + \int_V (\delta \mathbf{u})^T \rho \mathbf{b} dV = 0$$

$$- \int_V (\mathbf{L} \delta \mathbf{u})^T \mathbf{s} dV + \int_S (\delta \mathbf{u})^T \mathbf{t} dS + \int_V (\delta \mathbf{u})^T \rho \mathbf{b} dV = 0$$

$$\int_V (\mathbf{L} \delta \mathbf{u})^T \mathbf{s} dV = \int_S (\delta \mathbf{u})^T \mathbf{t} dS + \int_V (\delta \mathbf{u})^T \rho \mathbf{b} dV$$

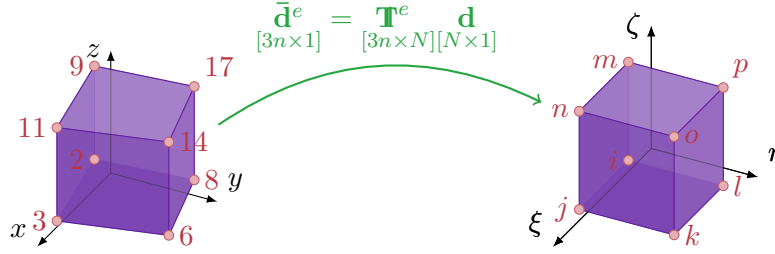
↑ praca sił wewnętrznych ↑ praca sił zewnętrznych

2.2. Dyskretyzacja MES ($n=LWE, N=LSSU, E=LEU$)

Aproksymacja pola przemieszczeń

$$\mathbf{u}^{eh} = \sum_{i=1}^n N_i^e(\xi, \eta, \zeta) \bar{\mathbf{d}}_i^e = \mathbf{N}^e \bar{\mathbf{d}}^e$$

$$\mathbf{N}^e_{[3 \times 3n]} = \begin{bmatrix} N_1^e & 0 & 0 & \dots & N_n^e & 0 & 0 \\ 0 & N_1^e & 0 & \dots & 0 & N_n^e & 0 \\ 0 & 0 & N_1^e & \dots & 0 & 0 & N_n^e \end{bmatrix} \quad \bar{\mathbf{d}}^e_{[3n \times 1]} = \begin{bmatrix} \bar{\mathbf{d}}_1^e \\ \dots \\ \bar{\mathbf{d}}_n^e \end{bmatrix}$$



\mathbf{T}^e – macierz transformacji uwzględniająca topologię i cosinusy kierunkowe pomiędzy osiami układu globalnego i lokalnego

2.3. Równanie równowagi układu zdyskretyzowanego

Równanie równowagi ($\rho \mathbf{b}^e = \mathbf{f}^e$ – wektor sił objętościowych)

$$\sum_{e=1}^E \left\{ \int_{V^e} (\mathbf{L}^e \delta \mathbf{u}^e)^T \mathbf{s}^e dV^e - \int_{S^e} (\delta \mathbf{u}^e)^T \mathbf{t}^e dS^e - \int_{V^e} (\delta \mathbf{u}^e)^T \mathbf{f}^e dV^e \right\} = 0$$

$$\sum_{e=1}^E \left\{ \int_{V^e} (\mathbf{L}^e \mathbf{N}^e \delta \bar{\mathbf{d}}^e)^T \mathbf{s}^e dV^e - \int_{S^e} (\mathbf{N}^e \delta \bar{\mathbf{d}}^e)^T \mathbf{t}^e dS^e - \int_{V^e} (\mathbf{N}^e \delta \bar{\mathbf{d}}^e)^T \mathbf{f}^e dV^e \right\} = 0$$

$$\sum_{e=1}^E (\delta \bar{\mathbf{d}}^e)^T \left\{ \int_{V^e} \mathbf{B}^{eT} \mathbf{s}^e dV^e - \int_{S^e} \mathbf{N}^{eT} \mathbf{t}^e dS^e - \int_{V^e} \mathbf{N}^{eT} \mathbf{f}^e dV^e \right\} = 0$$

$$(\delta \mathbf{d})^T \left\{ \sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{V^e} \mathbf{B}^{eT} \mathbf{s}^e dV^e - \int_{S^e} \mathbf{N}^{eT} \mathbf{t}^e dS^e - \int_{V^e} \mathbf{N}^{eT} \mathbf{f}^e dV^e \right\} \right\} = 0$$

$$\delta \mathbf{d} \neq \mathbf{0} \implies \sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{V^e} \mathbf{B}^{eT} \mathbf{s}^e dV^e - \int_{S^e} \mathbf{N}^{eT} \mathbf{t}^e dS^e - \int_{V^e} \mathbf{N}^{eT} \mathbf{f}^e dV^e \right\} = 0$$

$$\sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{V^e} \mathbf{B}^{eT} \mathbf{s}^e dV^e \right\} = \sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{S^e} \mathbf{N}^{eT} \mathbf{t}^e dS^e + \int_{V^e} \mathbf{N}^{eT} \mathbf{f}^e dV^e \right\}$$

siły wewnętrzne = siły zewnętrzne

Uwzględnienie związków kinematycznych i konstytutywnych

- liniowa sprężystość (zw. konstytutywny) : $\mathbf{s} = \mathbf{D}\mathbf{e}$
- liniowy związek kinematyczny : $\mathbf{e} = \mathbf{L}\mathbf{u}$

$$\mathbf{s}^e = \mathbf{D}^e \mathbf{L}^e \mathbf{u}^e = \mathbf{D}^e \mathbf{L}^e \mathbf{N}^e \bar{\mathbf{d}}^e = \mathbf{D}^e \mathbf{B}^e \mathbf{T}^e \mathbf{d}$$

$$\sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{V^e} \mathbf{B}^{eT} \mathbf{D}^e \mathbf{B}^e dV^e \right\} \mathbf{T}^e \mathbf{d} = \sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{S^e} \mathbf{N}^{eT} \mathbf{t}^e dS^e + \int_{V^e} \mathbf{N}^{eT} \mathbf{f}^e dV^e \right\}$$

$$\sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{V^e} \mathbf{B}^{eT} \mathbf{D}^e \mathbf{B}^e dV^e \right\} \mathbf{T}^e \mathbf{d} = \sum_{e=1}^E \mathbf{T}^{eT} \left\{ \int_{S^e} \mathbf{N}^{eT} \mathbf{t}^e dS^e + \int_{V^e} \mathbf{N}^{eT} \mathbf{f}^e dV^e \right\}$$

$\bar{\mathbf{K}}^e$ $\bar{\mathbf{p}}_b^e$ $\bar{\mathbf{p}}^e$

$$\sum_{e=1}^E \mathbf{T}^{eT} \bar{\mathbf{K}}^e \mathbf{T}^e \mathbf{d} = \sum_{e=1}^E \mathbf{T}^{eT} \bar{\mathbf{p}}_b^e + \sum_{e=1}^E \mathbf{T}^{eT} \bar{\mathbf{p}}^e$$

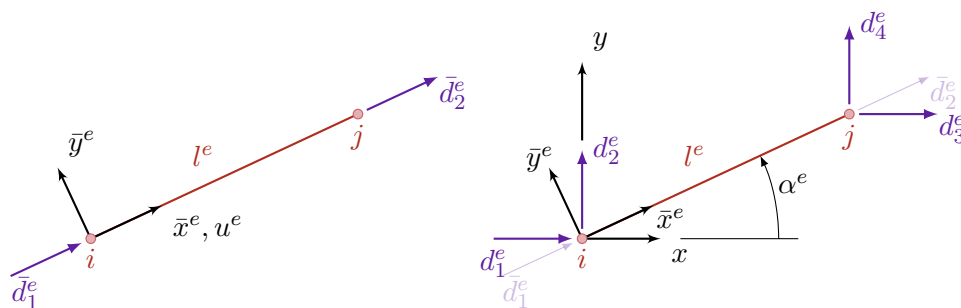
$$\sum_{e=1}^E \mathbf{T}^{eT} \bar{\mathbf{K}}^e \mathbf{T}^e \mathbf{d} = \sum_{e=1}^E \mathbf{T}^{eT} \bar{\mathbf{p}}_b^e + \sum_{e=1}^E \mathbf{T}^{eT} \bar{\mathbf{p}}^e$$

\mathbf{K} \mathbf{p}_b \mathbf{p}

$$\mathbf{K}\mathbf{d} = \mathbf{p}_b + \mathbf{p}$$

2.4. Elementy prętowe

2.4.1. Element kratowy



Wektor funkcji przemieszczeń

$$\mathbf{u} = \{u(x)\}$$

Wektor uogólnionych odkształceń

$$\mathbf{e} = \{\varepsilon_x\}$$

Wektor uogólnionych naprężeń

$$\mathbf{s} = \{N(x)\}$$

Wektor obciążenia po długości elementu

$$\mathbf{f} = \{f_x\}$$

Macierz związków konstytutywnych

$$\mathbf{D} = [EA]$$

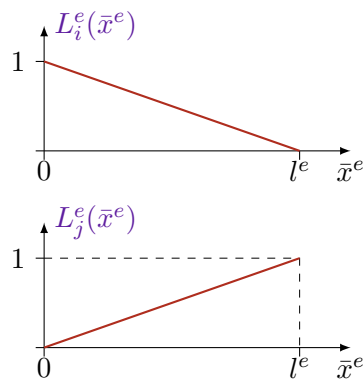
Macierz operatorów różniczkowych

$$\mathbf{L} = \left[\frac{d}{dx} \right]$$

Aproksymacja

$$\mathbf{u}^e(x) = \mathbf{N}^e(x)\bar{\mathbf{d}}^e$$

$$\mathbf{N}^e = [L_i^e \ L_j^e], \quad \bar{\mathbf{d}}^e = \begin{bmatrix} \bar{d}_1 \\ \bar{d}_2 \end{bmatrix}$$



Macierz sztywności

$$\bar{\mathbf{K}}^e = \int_0^{l^e} \mathbf{B}^{eT} \mathbf{D}^e \mathbf{B}^e d\bar{x}^e$$

$$\bar{\mathbf{K}}^e = \begin{bmatrix} \frac{EA}{l} & -\frac{EA}{l} \\ -\frac{EA}{l} & \frac{EA}{l} \end{bmatrix}^e$$

Macierz transformacji

$$c = \cos(\alpha^e) \quad \text{i} \quad s = \sin(\alpha^e)$$

$$\mathbf{T}^e = \begin{bmatrix} c & s & 0 & 0 \\ 0 & 0 & c & s \end{bmatrix}$$

Wzory transformacyjne

$$\mathbf{K}^e = \mathbf{T}^{eT} \bar{\mathbf{K}}^e \mathbf{T}^e$$

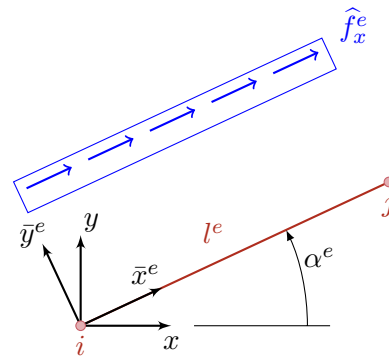
$$\mathbf{x}^e = \mathbf{T}^{eT} \bar{\mathbf{x}}^e, \quad \bar{\mathbf{x}}^e = \mathbf{T}^e \mathbf{x}^e$$

Wektor zastępników

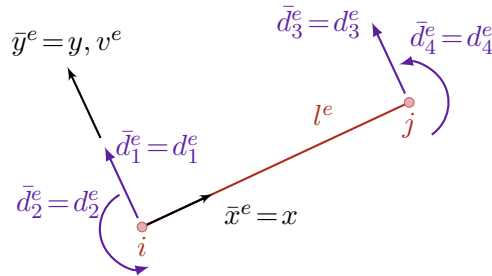
$$\bar{\mathbf{p}}^e = \int_0^{l^e} \mathbf{N}^{eT} f^e d\bar{x}^e$$

$$\text{dla } f_x = \text{const} = \hat{f}_x$$

$$\bar{\mathbf{p}}^e = \left\{ \frac{\hat{f}_x l}{2}, \frac{\hat{f}_x l}{2} \right\}^e$$



2.4.2. Element belkowy



Wektor funkcji przemieszczeń

$$\mathbf{u} = \{v(x)\}$$

Wektor uogólnionych odkształceń

$$\mathbf{e} = \{\kappa\}$$

Wektor uogólnionych naprężeń

$$\mathbf{s} = \{M(x)\}$$

Wektor obciążenia po długości elementu

$$\mathbf{f} = \{f_y\}$$

Macierz związków konstytutywnych

$$\mathbf{D} = \left[EI \right]$$

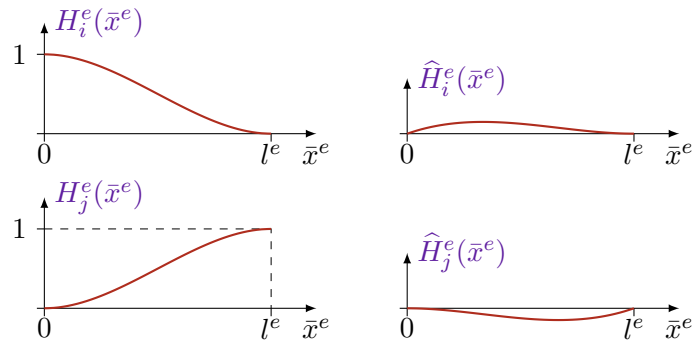
Macierz operatorów różniczkowych

$$\mathbf{L} = \left[-\frac{d^2}{dx^2} \right]$$

Aproksymacja

$$\mathbf{u}^e(x) = \mathbf{N}^e(x) \bar{\mathbf{d}}^e$$

$$\mathbf{N}^e = \begin{bmatrix} H_i^e & \hat{H}_i^e & H_j^e & \hat{H}_j^e \end{bmatrix}, \bar{\mathbf{d}}^e = \begin{bmatrix} \bar{d}_1 \\ \bar{d}_2 \\ \bar{d}_3 \\ \bar{d}_4 \end{bmatrix}$$



Macierz sztywności

$$\bar{\mathbf{K}}^e = \int_0^{l^e} \mathbf{B}^{eT} \mathbf{D}^e \mathbf{B}^e d\bar{x}^e$$

$$\bar{\mathbf{K}}^e = \frac{EI}{l^3} \begin{bmatrix} 12 & 6l & -12 & 6l \\ 6l & 4l^2 & -6l & 2l^2 \\ -12 & -6l & 12 & -6l \\ 6l & 2l^2 & -6l & 4l^2 \end{bmatrix}^e$$

Macierz transformacji

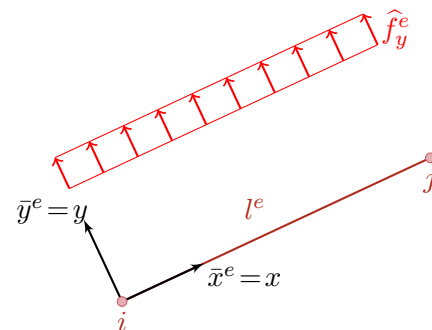
brak

Wektor zastępników

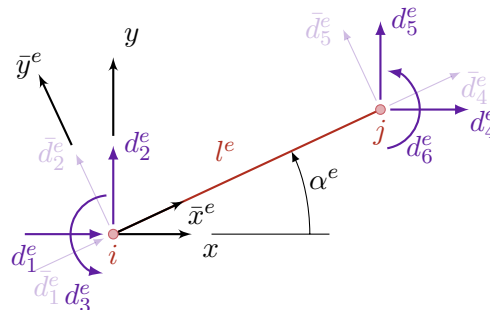
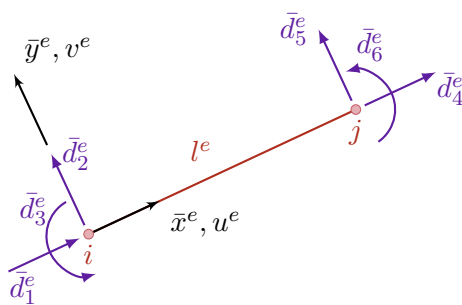
$$\bar{\mathbf{p}}^e = \int_0^{l^e} \mathbf{N}^{eT} f^e d\bar{x}^e$$

dla $f_y = \text{const} = \hat{f}_y$

$$\bar{\mathbf{p}}^e = \left\{ \frac{\hat{f}_y l}{2}, \frac{\hat{f}_y l^2}{12}, \frac{\hat{f}_y l}{2}, -\frac{\hat{f}_y l^2}{12} \right\}^e$$



2.4.3. Element ramowy



Wektor funkcji przemieszczeń

$$\mathbf{u} = \{u(x), v(x)\}$$

Wektor uogólnionych odkształceń

$$\mathbf{e} = \{\varepsilon_x, \kappa\}$$

Wektor uogólnionych naprężeń

$$\mathbf{s} = \{N(x), M(x)\}$$

Wektor obciążenia po długości elementu

$$\mathbf{f} = \{f_x, f_y\}$$

Macierz związków konstytutywnych

$$\mathbf{D} = \begin{bmatrix} EA & 0 \\ 0 & EI \end{bmatrix}$$

Macierz operatorów różniczkowych

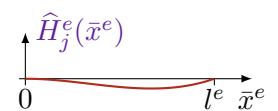
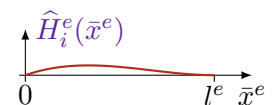
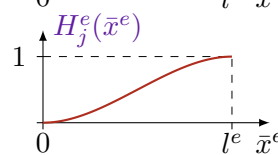
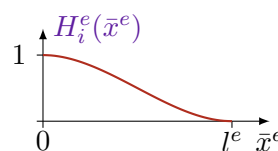
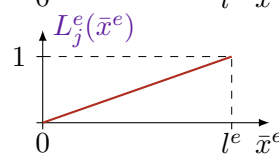
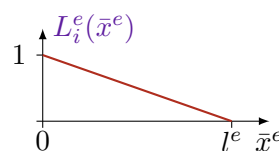
$$\mathbf{L} = \begin{bmatrix} \frac{d}{dx} & 0 \\ 0 & -\frac{d^2}{dx^2} \end{bmatrix}$$

Aproksymacja

$$\mathbf{u}^e(x) = \mathbf{N}^e(x) \bar{\mathbf{d}}^e$$

$$\mathbf{N}^e = \begin{bmatrix} L_i^e & 0 & 0 & L_j^e & 0 & 0 \\ 0 & H_i^e & \hat{H}_i^e & 0 & H_j^e & \hat{H}_j^e \end{bmatrix}$$

$$\bar{\mathbf{d}}^e = \begin{bmatrix} \bar{d}_1 \\ \bar{d}_2 \\ \bar{d}_3 \\ \bar{d}_4 \\ \bar{d}_5 \\ \bar{d}_6 \end{bmatrix}$$



Macierz sztywności

$$\bar{\mathbf{K}}^e = \int_0^{l^e} \mathbf{B}^{eT} \mathbf{D}^e \mathbf{B}^e d\bar{x}^e$$

$$\bar{\mathbf{K}}^e = \frac{EI}{l^3} \begin{bmatrix} \frac{Al^2}{I} & 0 & 0 & -\frac{Al^2}{I} & 0 & 0 \\ 0 & 12 & 6l & 0 & -12 & 6l \\ 0 & 6l & 4l^2 & 0 & -6l & 2l^2 \\ -\frac{Al^2}{I} & 0 & 0 & \frac{Al^2}{I} & 0 & 0 \\ 0 & -12 & -6l & 0 & 12 & -6l \\ 0 & 6l & 2l^2 & 0 & -6l & 4l^2 \end{bmatrix}^e$$

Macierz transformacji

$$c = \cos(\alpha^e) \text{ i } s = \sin(\alpha^e)$$

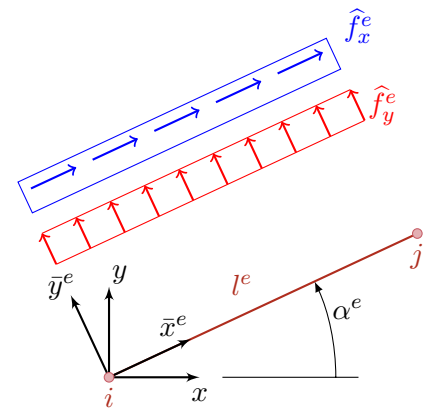
$$\mathbf{T}^e = \begin{bmatrix} c & s & 0 & 0 & 0 & 0 \\ -s & c & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & c & s & 0 \\ 0 & 0 & 0 & -s & c & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

Wektor zastępników

$$\bar{\mathbf{p}}^e = \int_0^{l^e} \mathbf{N}^{eT} \mathbf{f}^e d\bar{x}^e$$

$$\text{dla } f_x = \text{const} = \hat{f}_x \text{ i } f_y = \text{const} = \hat{f}_y$$

$$\bar{\mathbf{p}}^e = \left\{ \frac{\hat{f}_x l}{2}, \frac{\hat{f}_y l}{2}, \frac{\hat{f}_y l^2}{12}, \frac{\hat{f}_x l}{2}, \frac{\hat{f}_y l}{2}, -\frac{\hat{f}_y l^2}{12} \right\}^e$$

**2.5. Zagadnienie 2D****Wektor funkcji przemieszczeń**

$$\mathbf{u} = \{u(x, y), v(x, y)\}$$

Wektor odkształceń

$$\mathbf{e} = \{\varepsilon_x, \varepsilon_y, \gamma_{xy}\}$$

Wektor naprężeń

$$\mathbf{s} = \{\sigma_x, \sigma_y, \tau_{xy}\}$$

Wektor intensywności sił brzegowych

$$\mathbf{t} = \{t_x, t_y\}$$

Wektor intensywności sił powierzchniowych

$$\mathbf{f} = \{f_x, f_y\}$$

Macierz operatorów różniczkowych

$$\mathbf{L} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 \\ 0 & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix}$$

Macierz związków konstytutywnych PSN: $\sigma_z = 0 \quad \varepsilon_z = -\frac{\nu}{E}(\sigma_x + \sigma_y)$

$$\mathbf{D} = \frac{E}{1 - \nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix}$$

Macierz związków konstytutywnych PSO: $\varepsilon_z = 0 \quad \sigma_z = \nu(\sigma_x + \sigma_y)$

$$\mathbf{D} = \frac{E}{(1 + \nu)(1 - 2\nu)} \begin{bmatrix} 1 - \nu & \nu & 0 \\ \nu & 1 - \nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix}$$

Macierz sztywności (dla PSO $h^e = 1 \text{ m}$)

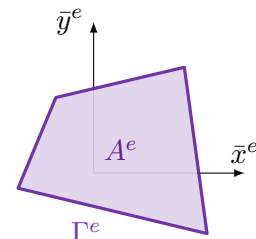
$$\bar{\mathbf{K}}^e = \int_{A^e} \mathbf{B}^{eT} \mathbf{D}^e \mathbf{B}^e h^e dA^e$$

Wektor obciążenia elementu (dla PSO $h^e = 1 \text{ m}$)

$$\bar{\mathbf{p}}^e = \int_{A^e} \mathbf{N}^{eT} \mathbf{f}^e h^e dA^e$$

Wektor sił brzegowych (dla PSO $h^e = 1 \text{ m}$)

$$\bar{\mathbf{p}}_b^e = \int_{\Gamma^e} \mathbf{N}^{eT} \mathbf{t}^e h^e d\Gamma^e$$

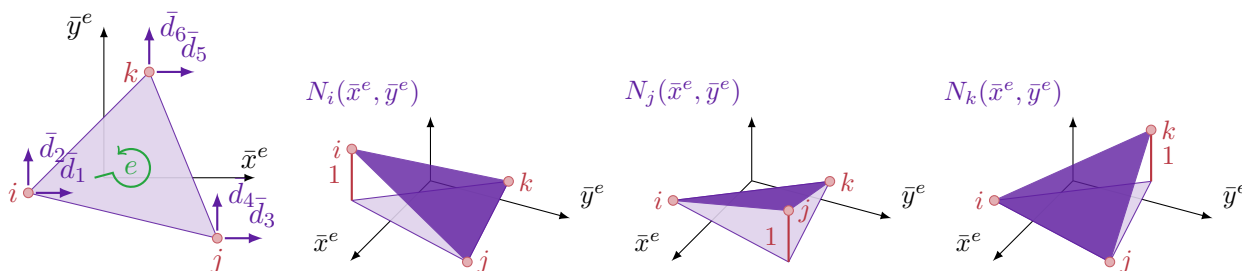


2.6. Elementy skończone 2D

Element trójwęzłowy

$$\mathbf{u}^e(x, y) = \mathbf{N}^e(x, y)\bar{\mathbf{d}}^e$$

$$\mathbf{N}^e = \begin{bmatrix} N_i^e & 0 & N_j^e & 0 & N_k^e & 0 \\ 0 & N_i^e & 0 & N_j^e & 0 & N_k^e \end{bmatrix}, \quad \bar{\mathbf{d}}^e = \begin{bmatrix} \bar{d}_1 \\ \bar{d}_2 \\ \bar{d}_3 \\ \bar{d}_4 \\ \bar{d}_5 \\ \bar{d}_6 \end{bmatrix}$$



Wyznaczenie funkcji kształtu dla elementu trójwęzłowego

Funkcja kształtu $N_i(\bar{x}^e, \bar{y}^e) \implies N_i(\bar{x}_i^e, \bar{y}_i^e) = 1, N_i(\bar{x}_j^e, \bar{y}_j^e) = 0, N_i(\bar{x}_k^e, \bar{y}_k^e) = 0$
 $N_i(\bar{x}^e, \bar{y}^e) = \alpha_{1i} + \alpha_{2i}\bar{x}^e + \alpha_{3i}\bar{y}^e$

$$\begin{bmatrix} 1 & \bar{x}_i & \bar{y}_i \\ 1 & \bar{x}_j & \bar{y}_j \\ 1 & \bar{x}_k & \bar{y}_k \end{bmatrix} \begin{bmatrix} \alpha_{1i} \\ \alpha_{2i} \\ \alpha_{3i} \end{bmatrix} = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

rozwiązanie układu równań metodą wyznaczników

$$W = \left| \begin{bmatrix} 1 & \bar{x}_i & \bar{y}_i \\ 1 & \bar{x}_j & \bar{y}_j \\ 1 & \bar{x}_k & \bar{y}_k \end{bmatrix} \right| = 2P_\Delta$$

$$W_{\alpha_{1i}} = \left| \begin{bmatrix} 1 & \bar{x}_i & \bar{y}_i \\ 0 & \bar{x}_j & \bar{y}_j \\ 0 & \bar{x}_k & \bar{y}_k \end{bmatrix} \right| = \bar{x}_j\bar{y}_k - \bar{x}_k\bar{y}_j \implies \alpha_{1i} = \frac{W_{\alpha_{1i}}}{W} = \frac{\bar{x}_j\bar{y}_k - \bar{x}_k\bar{y}_j}{2P_\Delta}$$

$$W_{\alpha_{2i}} = \left| \begin{bmatrix} 1 & 1 & \bar{y}_i \\ 1 & 0 & \bar{y}_j \\ 1 & 0 & \bar{y}_k \end{bmatrix} \right| = \bar{y}_j - \bar{y}_k, \implies \alpha_{2i} = \frac{W_{\alpha_{2i}}}{W} = \frac{\bar{y}_j - \bar{y}_k}{2P_\Delta}$$

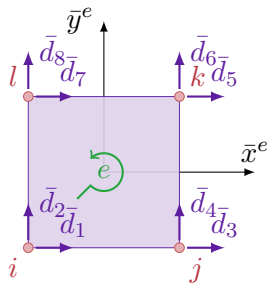
$$W_{\alpha_{3i}} = \left| \begin{bmatrix} 1 & \bar{x}_i & 1 \\ 1 & \bar{x}_j & 0 \\ 1 & \bar{x}_k & 0 \end{bmatrix} \right| = \bar{x}_k - \bar{x}_j \implies \alpha_{3i} = \frac{W_{\alpha_{3i}}}{W} = \frac{\bar{x}_k - \bar{x}_j}{2P_\Delta}$$

$$N_i(\bar{x}, \bar{y}) = \frac{\bar{x}_j\bar{y}_k - \bar{x}_k\bar{y}_j + (\bar{y}_j - \bar{y}_k)\bar{x}^e + (\bar{x}_k - \bar{x}_j)\bar{y}^e}{2P_\Delta}$$

Element czterowzłowy

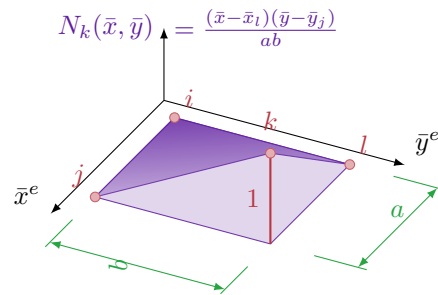
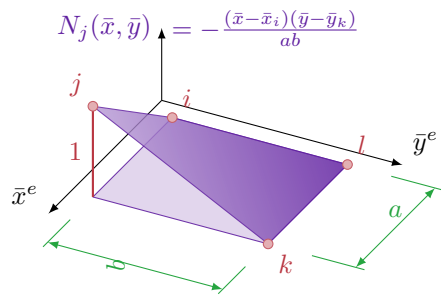
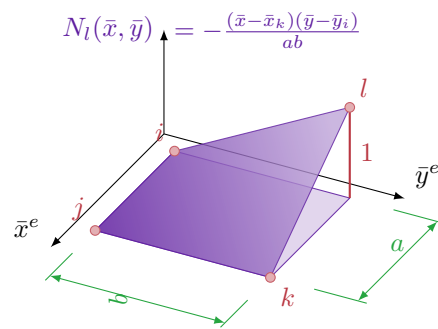
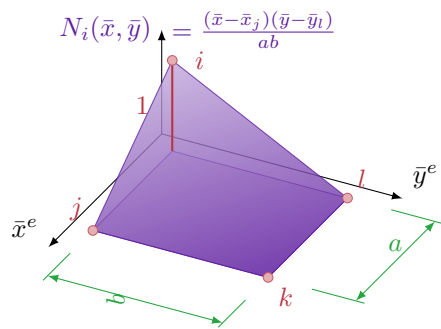
$$\mathbf{u}^e(\bar{x}, \bar{y}) = \mathbf{N}^e(\bar{x}, \bar{y})\bar{\mathbf{d}}^e$$

$$\mathbf{N}^e = \begin{bmatrix} N_i^e & 0 & N_j^e & 0 & N_k^e & 0 & N_l^e & 0 \\ 0 & N_i^e & 0 & N_j^e & 0 & N_k^e & 0 & N_l^e \end{bmatrix}, \quad \bar{\mathbf{d}}^e = \begin{bmatrix} \bar{d}_1 \\ \bar{d}_2 \\ \bar{d}_3 \\ \bar{d}_4 \\ \bar{d}_5 \\ \bar{d}_6 \\ \bar{d}_7 \\ \bar{d}_8 \end{bmatrix}$$



np. dla $N_i(\bar{x}^e, \bar{y}^e)$
 $N_i(\bar{x}_i^e, \bar{y}_i^e) = 1$
 $N_i(\bar{x}_j^e, \bar{y}_j^e) = 0$
 $N_i(\bar{x}_k^e, \bar{y}_k^e) = 0$
 $N_i(\bar{x}_l^e, \bar{y}_l^e) = 0$

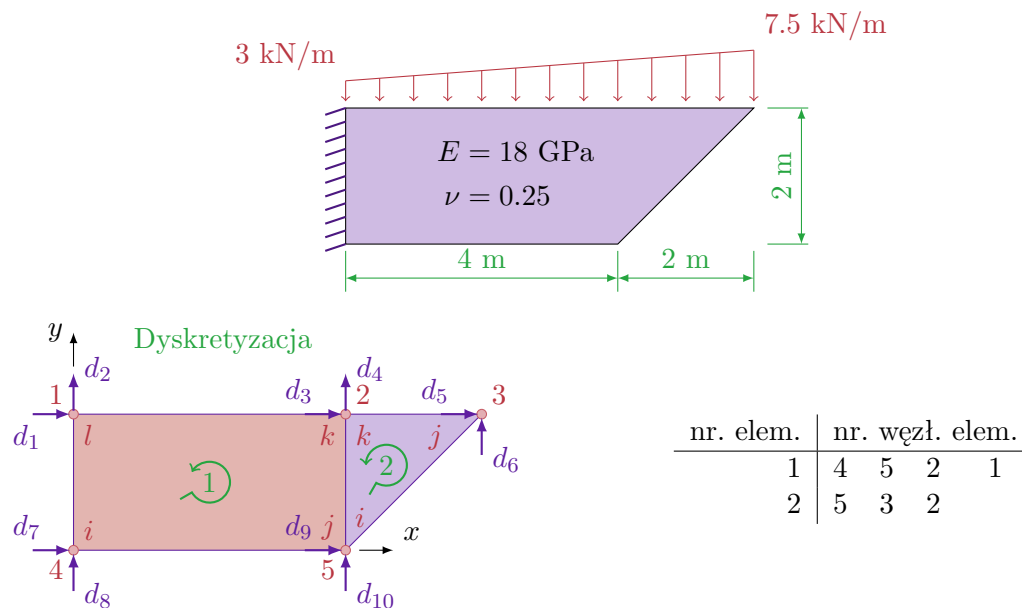
Element prostokątny



2.7. Przykłady

2.7.1. Zagadnienie PSO

1. Dane



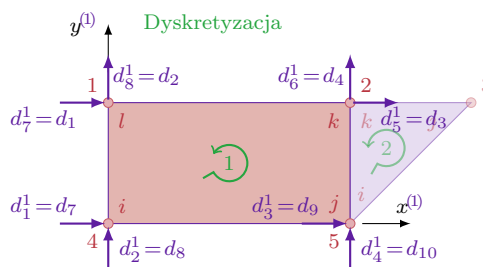
2. Macierz związków konstytutywnych

$$\mathbf{D} = \frac{18 \cdot 10^6}{(1 + 0.25)(1 - 2 \cdot 0.25)} \begin{bmatrix} 1 - 0.25 & 0.25 & 0 \\ 0.25 & 1 - 0.25 & 0 \\ 0 & 0 & \frac{1 - 2 \cdot 0.25}{2} \end{bmatrix} \text{ [kPa]}$$

$$\mathbf{D} = \begin{bmatrix} 21.6 & 7.2 & 0 \\ 7.2 & 21.6 & 0 \\ 0 & 0 & 7.2 \end{bmatrix} \cdot 10^6 \text{ [kPa]}$$

3. Wyznaczenie macierzy funkcji kształtu \mathbf{N}^e i macierzy sztywności \mathbf{K}^e

- Element 1



$$N_i^1(x^{(1)}, y^{(1)}) = \frac{x^{(1)}y^{(1)} - 2x^{(1)} - 4y^{(1)} + 8}{8}, \quad N_k^1(x^{(1)}, y^{(1)}) = \frac{x^{(1)}y^{(1)}}{8}$$

$$N_j^1(x^{(1)}, y^{(1)}) = \frac{2x^{(1)} - x^{(1)}y^{(1)}}{8}, \quad N_l^1(x^{(1)}, y^{(1)}) = \frac{4y^{(1)} - x^{(1)}y^{(1)}}{8}$$

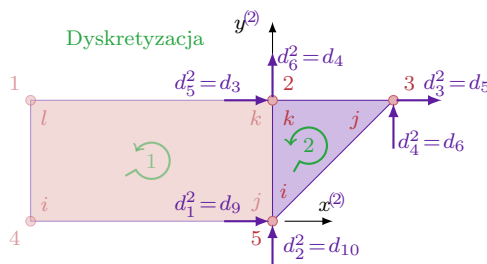
$$\mathbf{N}^1 = \begin{bmatrix} N_i^1 & 0 & N_j^1 & 0 & N_k^1 & 0 & N_l^1 & 0 \\ 0 & N_i^1 & 0 & N_j^1 & 0 & N_k^1 & 0 & N_l^1 \end{bmatrix}$$

$$\mathbf{B}^e(x^e, y^e) = \begin{bmatrix} \frac{dN_i^e}{dx} & 0 & \frac{dN_j^e}{dx} & 0 & \frac{dN_k^e}{dx} & 0 & \frac{dN_l^e}{dx} & 0 \\ 0 & \frac{dN_i^e}{dy} & 0 & \frac{dN_j^e}{dy} & 0 & \frac{dN_k^e}{dy} & 0 & \frac{dN_l^e}{dy} \\ \frac{dN_i^e}{dy} & \frac{dN_i^e}{dx} & \frac{dN_j^e}{dy} & \frac{dN_j^e}{dx} & \frac{dN_k^e}{dy} & \frac{dN_k^e}{dx} & \frac{dN_l^e}{dy} & \frac{dN_l^e}{dx} \end{bmatrix}$$

$$\mathbf{B}^1(x^{(1)}, y^{(1)}) = \begin{bmatrix} \frac{y^{(1)}}{8} - \frac{1}{4} & 0 & \frac{1}{4} - \frac{y^{(1)}}{8} & 0 & \frac{y^{(1)}}{8} & 0 & -\frac{y^{(1)}}{8} & 0 \\ 0 & \frac{x^{(1)}}{8} - \frac{1}{2} & 0 & -\frac{x^{(1)}}{8} & 0 & \frac{x^{(1)}}{8} & 0 & \frac{1}{2} - \frac{x^{(1)}}{8} \\ \frac{x^{(1)}}{8} - \frac{1}{2} & \frac{y^{(1)}}{8} - \frac{1}{4} & -\frac{x^{(1)}}{8} & \frac{1}{4} - \frac{y^{(1)}}{8} & \frac{x^{(1)}}{8} & \frac{y^{(1)}}{8} & \frac{1}{2} - \frac{x^{(1)}}{8} & -\frac{y^{(1)}}{8} \end{bmatrix}$$

$$\mathbf{K}^1 = \int_0^2 \int_0^4 \mathbf{B}^{1T} \mathbf{D} \mathbf{B}^1 dx^{(1)} dy^{(1)} = \begin{bmatrix} 84 & 36 & -12 & 0 & -42 & -36 & -30 & 0 \\ 36 & 156 & 0 & 60 & -36 & -78 & 0 & -138 \\ -12 & 0 & 84 & -36 & -30 & 0 & -42 & 36 \\ 0 & 60 & -36 & 156 & 0 & -138 & 36 & -78 \\ -42 & -36 & -30 & 0 & 84 & 36 & -12 & 0 \\ -36 & -78 & 0 & -138 & 36 & 156 & 0 & 60 \\ -30 & 0 & -42 & 36 & -12 & 0 & 84 & -36 \\ 0 & -138 & 36 & -78 & 0 & 60 & -36 & 156 \end{bmatrix} \cdot 10^5$$

- Element 2



$$N_i^2(x^{(2)}, y^{(2)}) = \frac{2 - y^{(2)}}{2}, \quad N_k^2(x^{(2)}, y^{(2)}) = \frac{y^{(2)} - x^{(2)}}{2}$$

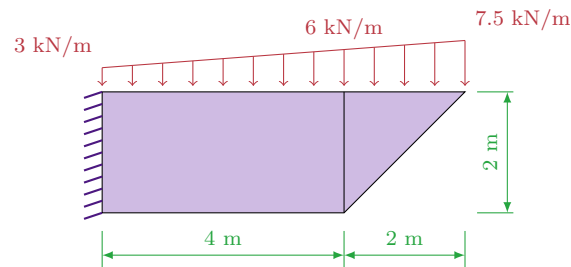
$$N_j^2(x^{(2)}, y^{(2)}) = \frac{x^{(2)}}{2}$$

$$\mathbf{N}^2 = \begin{bmatrix} N_i^2 & 0 & N_j^2 & 0 & N_k^2 & 0 \\ 0 & N_i^2 & 0 & N_j^2 & 0 & N_k^2 \end{bmatrix}$$

$$\mathbf{B}^e(x^e, y^e) = \begin{bmatrix} \frac{dN_i^e}{dx} & 0 & \frac{dN_j^e}{dx} & 0 & \frac{dN_k^e}{dx} & 0 \\ 0 & \frac{dN_i^e}{dy} & 0 & \frac{dN_j^e}{dy} & 0 & \frac{dN_k^e}{dy} \\ \frac{dN_i^e}{dy} & \frac{dN_i^e}{dx} & \frac{dN_j^e}{dy} & \frac{dN_j^e}{dx} & \frac{dN_k^e}{dy} & \frac{dN_k^e}{dx} \end{bmatrix}$$

$$\mathbf{B}^2(x^{(2)}, y^{(2)}) = \begin{bmatrix} 0 & 0 & \frac{1}{2} & 0 & -\frac{1}{2} & 0 \\ 0 & -\frac{1}{2} & 0 & 0 & 0 & \frac{1}{2} \\ -\frac{1}{2} & 0 & 0 & \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} \end{bmatrix}$$

$$\mathbf{K}^2 = \mathbf{B}^{2T} \mathbf{D} \mathbf{B}^2 A^2 = \begin{bmatrix} 36 & 0 & 0 & -36 & -36 & 36 \\ 0 & 108 & -36 & 0 & 36 & -108 \\ 0 & -36 & 108 & 0 & -108 & 36 \\ -36 & 0 & 0 & 36 & 36 & -36 \\ -36 & 36 & -108 & 36 & 144 & -72 \\ 36 & -108 & 36 & -36 & -72 & 144 \end{bmatrix} \cdot 10^5$$

4. Wektor \mathbf{p}_b 

• Element 1

$$\mathbf{p}_b^1 = \int_{\Gamma_{ij}^1} \mathbf{N}^{1T} \overset{w.b. = 0}{\mathbf{t}} d\Gamma + \int_{\Gamma_{jk}^1} \mathbf{N}^{1T} \mathbf{t} d\Gamma + \int_{\Gamma_{kl}^1} \mathbf{N}^{1T} \mathbf{t} d\Gamma + \int_{\Gamma_{li}^1} \mathbf{N}^{1T} \mathbf{t} d\Gamma$$

wspólna krawędź – równowaga sił
wzdłuż linii 2-5: $\mathbf{t}_{jk}^1 = -\mathbf{t}_{ki}^2$

$$= \int_0^4 \left(\mathbf{N}^1(x^{(1)}, y^{(1)}=2) \right)^T \left[-3 \left(1 - \frac{x^{(1)}}{4} \right) - 6 \frac{x^{(1)}}{4} \right] dx^{(1)}$$

$$+ \int_0^2 \left(\mathbf{N}^1(x^{(1)}=0, y^{(1)}) \right)^T \mathbf{t} dy^{(1)}$$

$$= \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ -10 \\ 0 \\ -8 \end{bmatrix} + \begin{bmatrix} r_1^1 \\ r_2^1 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_7^1 \\ r_8^1 \end{bmatrix}$$

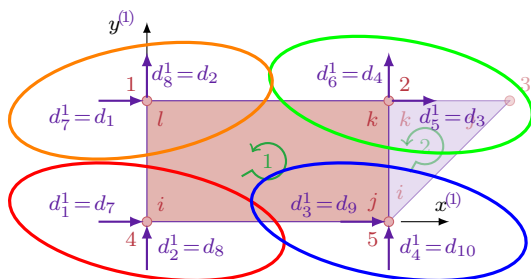
- Element 2

wspólna krawędź – równowaga sił
wzdłuż linii 2-5: $t_{jk}^1 = -t_{ki}^2$

$$\begin{aligned}
 \mathbf{p}_b^2 &= \int_{\Gamma_{ij}^2} \mathbf{N}^{2T} \mathbf{t} \, d\Gamma + \int_{\Gamma_{jk}^2} \mathbf{N}^{2T} \mathbf{t} \, d\Gamma + \int_{\Gamma_{ki}^2} \mathbf{N}^{2T} \mathbf{t} \, d\Gamma \\
 &= \int_0^2 \left(\mathbf{N}^2(x^{(2)}, y^{(2)}=2) \right)^T \begin{bmatrix} 0 \\ -6 \left(1 - \frac{x^{(2)}}{2} \right) - 7.5 \frac{x^{(2)}}{2} \\ 0 \end{bmatrix} dx^{(2)} \\
 &= \begin{bmatrix} 0 \\ 0 \\ 0 \\ -7 \\ 0 \\ -6.5 \end{bmatrix}
 \end{aligned}$$

5. Agregacja – Macierze Boole’a \mathbf{B}

- Element 1

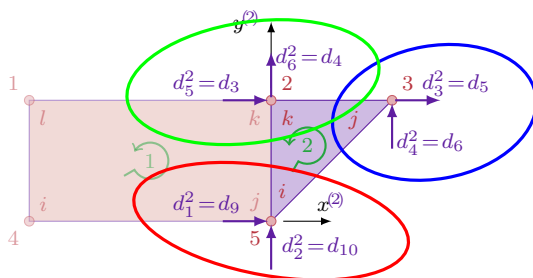


$$\mathbf{B}^1 = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

nr. lok. (1) (2) (3) (4) (5) (6) (7) (8)

nr. glob. (1) (2) (3) (4) (5) (6) (7) (8) (9) (10)

- Element 2



$$\mathbf{B}^2 = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

nr. lok. (1) (2) (3) (4) (5) (6)

nr. glob. (1) (2) (3) (4) (5) (6) (7) (8) (9) (10)

6. Agregacja - Macierz sztywności

$$\mathbf{K} = \mathbf{B}^{1T} \mathbf{K}^1 \mathbf{B}^1 + \mathbf{B}^{2T} \mathbf{K}^2 \mathbf{B}^2$$

$$\mathbf{K} = \begin{bmatrix} 84 & -36 & -12 & 0 & 0 & 0 & -30 & 0 & -42 & 36 \\ -36 & 156 & 0 & 60 & 0 & 0 & 0 & -138 & 36 & -78 \\ -12 & 0 & 228 & -36 & -108 & 36 & -42 & -36 & -66 & 36 \\ 0 & 60 & -36 & 300 & 36 & -36 & -36 & -78 & 36 & -246 \\ 0 & 0 & -108 & 36 & 108 & 0 & 0 & 0 & 0 & -36 \\ 0 & 0 & 36 & -36 & 0 & 36 & 0 & 0 & -36 & 0 \\ -30 & 0 & -42 & -36 & 0 & 0 & 84 & 36 & -12 & 0 \\ 0 & -138 & -36 & -78 & 0 & 0 & 36 & 156 & 0 & 60 \\ -42 & 36 & -66 & 36 & 0 & -36 & -12 & 0 & 120 & -36 \\ 36 & -78 & 36 & -246 & -36 & 0 & 0 & 60 & -36 & 264 \end{bmatrix} \cdot 10^5$$

7. Agregacja - Wektor obciążenia

$$\mathbf{p}_b = \mathbf{B}^{1T} \mathbf{p}_b^1 + \mathbf{B}^{2T} \mathbf{p}_b^2, \quad \mathbf{p} = \mathbf{0}$$

$$\mathbf{p}_b = \begin{bmatrix} 0 \\ -8 \\ 0 \\ -16.5 \\ 0 \\ -7 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} r_7^1 = r_1 \\ r_8^1 = r_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_1^1 = r_7 \\ r_2^1 = r_8 \\ 0 \\ 0 \end{bmatrix}$$

8. Układ równań MES: $\mathbf{Kd} = \mathbf{p} + \mathbf{p}_b$

$$\begin{bmatrix} 84 & -36 & -12 & 0 & 0 & 0 & -30 & 0 & -42 & 36 \\ -36 & 156 & 0 & 60 & 0 & 0 & 0 & -138 & 36 & -78 \\ -12 & 0 & 228 & -36 & -108 & 36 & -42 & -36 & -66 & 36 \\ 0 & 60 & -36 & 300 & 36 & -36 & -36 & -78 & 36 & -246 \\ 0 & 0 & -108 & 36 & 108 & 0 & 0 & 0 & 0 & -36 \\ 0 & 0 & 36 & -36 & 0 & 36 & 0 & 0 & -36 & 0 \\ -30 & 0 & -42 & -36 & 0 & 0 & 84 & 36 & -12 & 0 \\ 0 & -138 & -36 & -78 & 0 & 0 & 36 & 156 & 0 & 60 \\ -42 & 36 & -66 & 36 & 0 & -36 & -12 & 0 & 120 & -36 \\ 36 & -78 & 36 & -246 & -36 & 0 & 0 & 60 & -36 & 264 \end{bmatrix} \cdot 10^5 \begin{bmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \\ d_7 \\ d_8 \\ d_9 \\ d_{10} \end{bmatrix} = \begin{bmatrix} 0 \\ -8 \\ 0 \\ -16.5 \\ 0 \\ -7 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} r_1 \\ r_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_7 \\ r_8 \\ 0 \\ 0 \end{bmatrix}$$

9. Uwzględnienie warunków brzegowych

$$\begin{bmatrix} 84 & -36 & -12 & 0 & 0 & 0 & -30 & 0 & -42 & 36 \\ -36 & 156 & 0 & 60 & 0 & 0 & 0 & -138 & 36 & -78 \\ -12 & 0 & 228 & -36 & -108 & 36 & -42 & -36 & -66 & 36 \\ 0 & 60 & -36 & 300 & 36 & -36 & -36 & -78 & 36 & -246 \\ 0 & 0 & -108 & 36 & 108 & 0 & 0 & 0 & 0 & -36 \\ 0 & 0 & 36 & -36 & 0 & 36 & 0 & 0 & -36 & 0 \\ -30 & 0 & -42 & -36 & 0 & 0 & 84 & 36 & -12 & 0 \\ 0 & -138 & -36 & -78 & 0 & 0 & 36 & 156 & 0 & 60 \\ -42 & 36 & -66 & 36 & 0 & -36 & -12 & 0 & 120 & -36 \\ 36 & -78 & 36 & -246 & -36 & 0 & 0 & 60 & -36 & 264 \end{bmatrix} \cdot 10^5 \begin{bmatrix} 0 \\ 0 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \\ 0 \\ 0 \\ d_9 \\ d_{10} \end{bmatrix} = \begin{bmatrix} 0 \\ -8 \\ 0 \\ -16.5 \\ 0 \\ -7 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} r_1 \\ r_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_7 \\ r_8 \\ 0 \\ 0 \end{bmatrix}$$

Rozwiązanie:

$$\mathbf{d} = \{0 \ 0 \ 0.743 \ -2.131 \ 0.760 \ -3.777 \ 0 \ 0 \ -0.709 \ -2.080\} \cdot 10^{-5} \text{ m}$$

$$\mathbf{r} = \{-54 \ 16.851 \ 0 \ 0 \ 0 \ 0 \ 54 \ 14.649 \ 0 \ 0\} \text{ kN}$$

10. Powrót do elementu

- Element 1

$$\mathbf{d}^1 = \mathbf{B}^1 \mathbf{d} = \{0 \ 0 \ -0.709 \ -2.080 \ 0.743 \ -2.131 \ 0 \ 0\} \cdot 10^{-5}$$

$$\mathbf{e}^1 = \mathbf{B}^1 \mathbf{d}^1$$

$$\mathbf{e}^1 = \begin{bmatrix} 1.815y - 1.773 \\ -0.064x \\ 1.815x - 0.064y - 5.199 \end{bmatrix} \cdot 10^{-6}, \quad \mathbf{e}^1(2,1) = \begin{bmatrix} 0.424 \\ -1.272 \\ -16.319 \end{bmatrix} \cdot 10^{-7}$$

$$\mathbf{s}^1 = \mathbf{D} \mathbf{e}^1$$

$$\mathbf{s}^1 = \begin{bmatrix} 39.214y - 0.458x - 38.299 \\ 13.071y - 1.373x - 12.766 \\ 13.071x - 0.458y - 37.435 \end{bmatrix}, \quad \mathbf{s}^1(2,1) = \begin{bmatrix} 0 \\ -2.442 \\ -11.750 \end{bmatrix} \text{ kPa}$$

$$\sigma_z^1 = \nu(\sigma_x^1 + \sigma_y^1) = 0.25(0 - 2.442) = -0.615 \text{ kPa}$$

- Element 2

$$\mathbf{d}^2 = \mathbf{B}^2 \mathbf{d} = \{-0.709 \ -2.080 \ 0.760 \ -3.777 \ 0.743 \ -2.131\} \cdot 10^{-5}$$

$$\mathbf{e}^2 = \mathbf{B}^2 \mathbf{d}^2$$

$$\mathbf{e}^2 = \begin{bmatrix} 0.848 \\ -2.543 \\ -9.722 \end{bmatrix} \cdot 10^{-7}$$

$$\mathbf{s}^2 = \begin{bmatrix} 0 \\ -4.883 \\ -7.000 \end{bmatrix} \text{ kPa}, \quad \sigma_z^2 = \nu(\sigma_x^2 + \sigma_y^2) = 0.25(0 - 4.883) = -1.221 \text{ kPa}$$

11. Wyznaczenie wartości przemieszczeń w wybranych punktach

- Element 1 – punkt $(x^{(1)} = 2, y^{(1)} = 1)$

$$\mathbf{u}^1(x^{(1)}, y^{(1)}) = \mathbf{N}^1(x^{(1)}, y^{(1)}) \mathbf{d}^1$$

$$\mathbf{u}^1(2, 1) = \begin{bmatrix} \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 \\ 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} \end{bmatrix} \cdot \begin{bmatrix} 0 \\ 0 \\ -0.709 \\ -2.080 \\ 0.743 \\ -2.131 \\ 0 \\ 0 \end{bmatrix} \cdot 10^{-5} = \begin{bmatrix} 0.0085 \\ -1.0526 \end{bmatrix} \cdot 10^{-5} \text{ [m]}$$

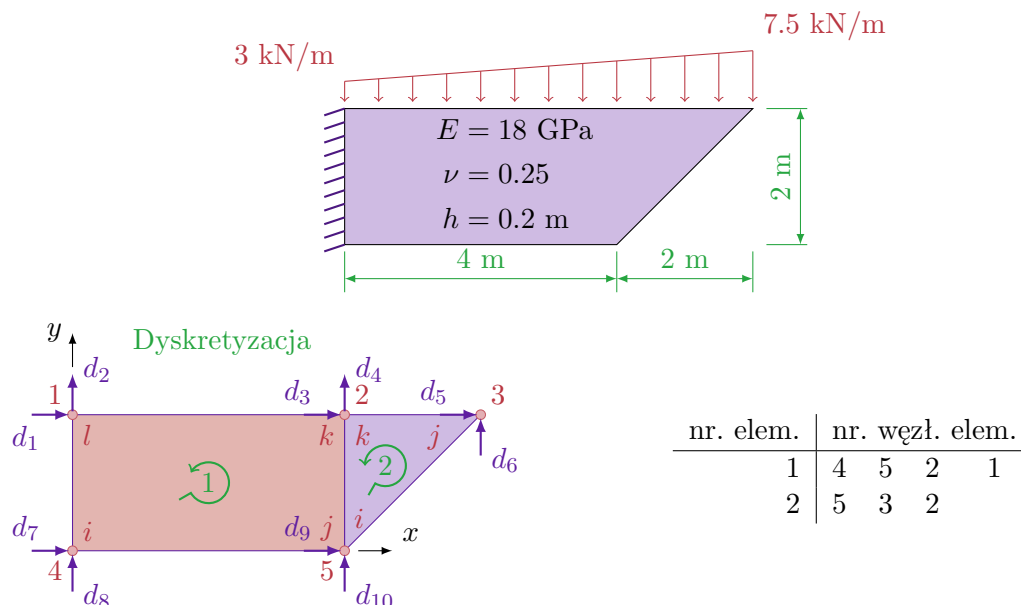
- Element 2 – punkt $(x^{(2)} = 0.5, y^{(2)} = 1.5)$

$$\mathbf{u}^2(x^{(2)}, y^{(2)}) = \mathbf{N}^2(x^{(2)}, y^{(2)}) \mathbf{d}^2$$

$$\mathbf{u}^2(0.5, 1.5) = \begin{bmatrix} \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{2} & 0 \\ 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{2} \end{bmatrix} \cdot \begin{bmatrix} -0.709 \\ -2.080 \\ 0.760 \\ -3.777 \\ 0.743 \\ -2.131 \end{bmatrix} \cdot 10^{-5} = \begin{bmatrix} 0.3843 \\ -2.5296 \end{bmatrix} \cdot 10^{-5} \text{ [m]}$$

2.7.2. Statyka tarczy

1. Dane



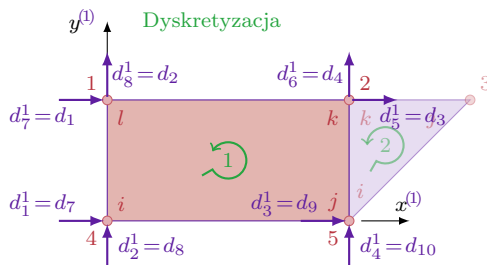
2. Macierz związków konstytutywnych

$$\mathbf{D} = \frac{18 \cdot 10^6}{1 - 0.25^2} \begin{bmatrix} 1 & 0.25 & 0 \\ 0.25 & 1 & 0 \\ 0 & 0 & \frac{1-0.25}{2} \end{bmatrix} \text{ [kPa]}$$

$$\mathbf{D} = \begin{bmatrix} 19.2 & 4.8 & 0 \\ 4.8 & 19.2 & 0 \\ 0 & 0 & 7.2 \end{bmatrix} \cdot 10^6 \text{ [kPa]}$$

3. Wyznaczenie macierzy funkcji kształtu \mathbf{N}^e i macierzy sztywności \mathbf{K}^e

- Element 1



$$N_i^1(x^{(1)}, y^{(1)}) = \frac{x^{(1)}y^{(1)} - 2x^{(1)} - 4y^{(1)} + 8}{8}, \quad N_k^1(x^{(1)}, y^{(1)}) = \frac{x^{(1)}y^{(1)}}{8}$$

$$N_j^1(x^{(1)}, y^{(1)}) = \frac{2x^{(1)} - x^{(1)}y^{(1)}}{8}, \quad N_l^1(x^{(1)}, y^{(1)}) = \frac{4y^{(1)} - x^{(1)}y^{(1)}}{8}$$

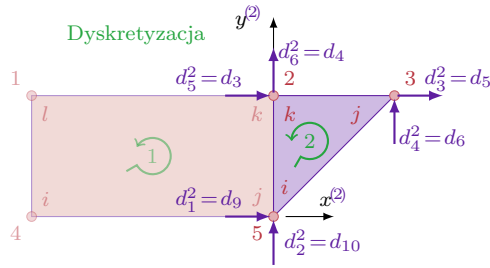
$$\mathbf{N}^1 = \begin{bmatrix} N_i^1 & 0 & N_j^1 & 0 & N_k^1 & 0 & N_l^1 & 0 \\ 0 & N_i^1 & 0 & N_j^1 & 0 & N_k^1 & 0 & N_l^1 \end{bmatrix}$$

$$\mathbf{B}^e(x^e, y^e) = \begin{bmatrix} \frac{dN_i^e}{dx} & 0 & \frac{dN_j^e}{dx} & 0 & \frac{dN_k^e}{dx} & 0 & \frac{dN_l^e}{dx} & 0 \\ 0 & \frac{dN_i^e}{dy} & 0 & \frac{dN_j^e}{dy} & 0 & \frac{dN_k^e}{dy} & 0 & \frac{dN_l^e}{dy} \\ \frac{dN_i^e}{dy} & \frac{dN_i^e}{dx} & \frac{dN_j^e}{dy} & \frac{dN_j^e}{dx} & \frac{dN_k^e}{dy} & \frac{dN_k^e}{dx} & \frac{dN_l^e}{dy} & \frac{dN_l^e}{dx} \end{bmatrix}$$

$$\mathbf{B}^1(x^{(1)}, y^{(1)}) = \begin{bmatrix} \frac{y^{(1)}}{8} - \frac{1}{4} & 0 & \frac{1}{4} - \frac{y^{(1)}}{8} & 0 & \frac{y^{(1)}}{8} & 0 & -\frac{y^{(1)}}{8} & 0 \\ 0 & \frac{x^{(1)}}{8} - \frac{1}{2} & 0 & -\frac{x^{(1)}}{8} & 0 & \frac{x^{(1)}}{8} & 0 & \frac{1}{2} - \frac{x^{(1)}}{8} \\ \frac{x^{(1)}}{8} - \frac{1}{2} & \frac{y^{(1)}}{8} - \frac{1}{4} & -\frac{x^{(1)}}{8} & \frac{1}{4} - \frac{y^{(1)}}{8} & \frac{x^{(1)}}{8} & \frac{y^{(1)}}{8} & \frac{1}{2} - \frac{x^{(1)}}{8} & -\frac{y^{(1)}}{8} \end{bmatrix}$$

$$\mathbf{K}^1 = \int_0^2 \int_0^4 \mathbf{B}^{1T} \mathbf{D} \mathbf{B}^1 h \, dx^{(1)} dy^{(1)} = \begin{bmatrix} 16 & 6 & -1.6 & -1.2 & -8 & -6 & -6.4 & 1.2 \\ 6 & 28 & 1.2 & 10.4 & -6 & -14 & -1.2 & -24.4 \\ -1.6 & 1.2 & 16 & -6 & -6.4 & -1.2 & -8 & 6 \\ -1.2 & 10.4 & -6 & 28 & 1.2 & -24.4 & 6 & -14 \\ -8 & -6 & -6.4 & 1.2 & 16 & 6 & -1.6 & -1.2 \\ -6 & -14 & -1.2 & -24.4 & 6 & 28 & 1.2 & 10.4 \\ -6.4 & -1.2 & -8 & 6 & -1.6 & 1.2 & 16 & -6 \\ 1.2 & -24.4 & 6 & -14 & -1.2 & 10.4 & -6 & 28 \end{bmatrix} \cdot 10^5$$

- Element 2



$$N_i^2(x^{(2)}, y^{(2)}) = \frac{2 - y^{(2)}}{2}, \quad N_k^2(x^{(2)}, y^{(2)}) = \frac{y^{(2)} - x^{(2)}}{2}$$

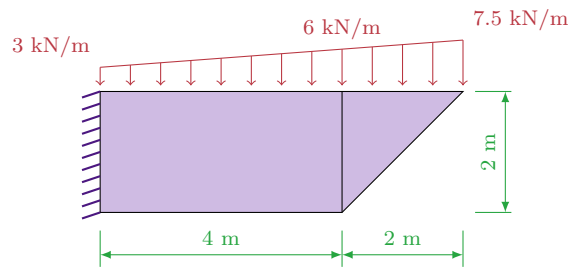
$$N_j^2(x^{(2)}, y^{(2)}) = \frac{x^{(2)}}{2}$$

$$\mathbf{N}^2 = \begin{bmatrix} N_i^2 & 0 & N_j^2 & 0 & N_k^2 & 0 \\ 0 & N_i^2 & 0 & N_j^2 & 0 & N_k^2 \end{bmatrix}$$

$$\mathbf{B}^e(x^e, y^e) = \begin{bmatrix} \frac{dN_i^e}{dx} & 0 & \frac{dN_j^e}{dx} & 0 & \frac{dN_k^e}{dx} & 0 \\ 0 & \frac{dN_i^e}{dy} & 0 & \frac{dN_j^e}{dy} & 0 & \frac{dN_k^e}{dy} \\ \frac{dN_i^e}{dy} & \frac{dN_i^e}{dx} & \frac{dN_j^e}{dy} & \frac{dN_j^e}{dx} & \frac{dN_k^e}{dy} & \frac{dN_k^e}{dx} \end{bmatrix}$$

$$\mathbf{B}^2(x^{(2)}, y^{(2)}) = \begin{bmatrix} 0 & 0 & \frac{1}{2} & 0 & -\frac{1}{2} & 0 \\ 0 & -\frac{1}{2} & 0 & 0 & 0 & \frac{1}{2} \\ -\frac{1}{2} & 0 & 0 & \frac{1}{2} & \frac{1}{2} & -\frac{1}{2} \end{bmatrix}$$

$$\mathbf{K}^2 = \mathbf{B}^{2T} \mathbf{D} \mathbf{B}^2 h A^2 = \begin{bmatrix} 7.2 & 0 & 0 & -7.2 & -7.2 & 7.2 \\ 0 & 19.2 & -4.8 & 0 & 4.8 & -19.2 \\ 0 & -4.8 & 19.2 & 0 & -19.2 & 4.8 \\ -7.2 & 0 & 0 & 7.2 & 7.2 & -7.2 \\ -7.2 & 4.8 & -19.2 & 7.2 & 26.4 & -12 \\ 7.2 & -19.2 & 4.8 & -7.2 & -12 & 26.4 \end{bmatrix} \cdot 10^5$$

4. Wektor \mathbf{p}_b 

- Element 1

$$\begin{aligned}
 \mathbf{p}_b^1 &= \int_{\Gamma_{ij}^1} \mathbf{N}^{1T} \mathbf{t} \, d\Gamma + \int_{\Gamma_{jk}^1} \mathbf{N}^{1T} \mathbf{t} \, d\Gamma + \int_{\Gamma_{kl}^1} \mathbf{N}^{1T} \mathbf{t} \, d\Gamma + \int_{\Gamma_{li}^1} \mathbf{N}^{1T} \mathbf{t} \, d\Gamma \\
 &= \int_0^4 \left(\mathbf{N}^1(x^{(1)}, y^{(1)}=2) \right)^T \left[-3 \left(1 - \frac{x^{(1)}}{4} \right) - 6 \frac{x^{(1)}}{4} \right] dx^{(1)} \\
 &\quad + \int_0^2 \left(\mathbf{N}^1(x^{(1)}=0, y^{(1)}) \right)^T \mathbf{t} dy^{(1)} \\
 &= \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ -10 \\ 0 \\ -8 \end{bmatrix} + \begin{bmatrix} r_1^1 \\ r_2^1 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_7^1 \\ r_8^1 \end{bmatrix}
 \end{aligned}$$

wspólna krawędź – równowaga sił
wzdłuż linii 2-5: $\mathbf{t}_{jk}^1 = -\mathbf{t}_{ki}^2$

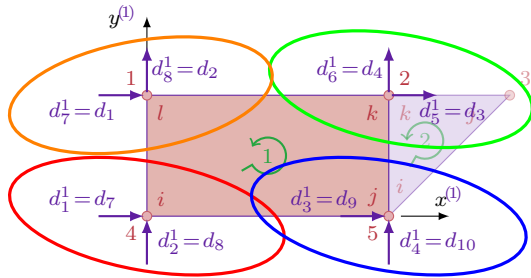
- Element 2

$$\begin{aligned}
 \mathbf{p}_b^2 &= \int_{\Gamma_{ij}^2} \mathbf{N}^{2T} \mathbf{t} \, d\Gamma + \int_{\Gamma_{jk}^2} \mathbf{N}^{2T} \mathbf{t} \, d\Gamma + \int_{\Gamma_{ki}^2} \mathbf{N}^{2T} \mathbf{t} \, d\Gamma \\
 &= \int_0^2 \left(\mathbf{N}^2(x^{(2)}, y^{(2)}=2) \right)^T \left[-6 \left(1 - \frac{x^{(2)}}{2} \right) - 7.5 \frac{x^{(2)}}{2} \right] dx^{(2)} \\
 &= \begin{bmatrix} 0 \\ 0 \\ 0 \\ -7 \\ 0 \\ -6.5 \end{bmatrix}
 \end{aligned}$$

wspólna krawędź – równowaga sił
wzdłuż linii 2-5: $\mathbf{t}_{jk}^1 = -\mathbf{t}_{ki}^2$

5. Agregacja – Macierze Boole’a \mathbf{B}

- Element 1

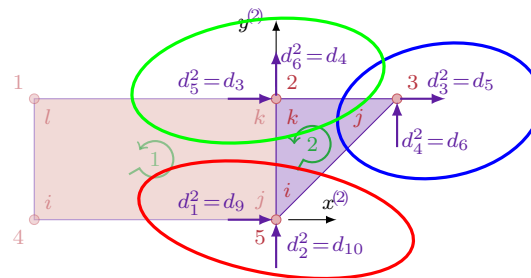


$$\mathbf{B}^1 = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

nr. lok. 1 2 3 4 5 6 7 8 9 10

nr. glob. 1 2 3 4 5 6 7 8 9 10

- Element 2



$$\mathbf{B}^2 = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix}$$

nr. lok. 1 2 3 4 5 6 7 8 9 10

nr. glob. 1 2 3 4 5 6 7 8 9 10

6. Agregacja - Macierz sztywności

$$\mathbf{K} = \mathbf{B}^{1T} \mathbf{K}^1 \mathbf{B}^1 + \mathbf{B}^{2T} \mathbf{K}^2 \mathbf{B}^2$$

$$\mathbf{K} = \begin{bmatrix} 16 & -6 & -1.6 & 1.2 & 0 & 0 & -6.4 & -1.2 & -8 & 6 \\ -6 & 28 & -1.2 & 10.4 & 0 & 0 & 1.2 & -24.4 & 6 & -14 \\ -1.6 & -1.2 & 42.4 & -6 & -19.2 & 7.2 & -8 & -6 & -13.6 & 6 \\ 1.2 & 10.4 & -6 & 54.4 & 4.8 & -7.2 & -6 & -14 & 6 & -43.6 \\ 0 & 0 & -19.2 & 4.8 & 19.2 & 0 & 0 & 0 & 0 & -4.8 \\ 0 & 0 & 7.2 & -7.2 & 0 & 7.2 & 0 & 0 & -7.2 & 0 \\ -6.4 & 1.2 & -8 & -6 & 0 & 0 & 16 & 6 & -1.6 & -1.2 \\ -1.2 & -24.4 & -6 & -14 & 0 & 0 & 6 & 28 & 1.2 & 10.4 \\ -8 & 6 & -13.6 & 6 & 0 & -7.2 & -1.6 & 1.2 & 23.2 & -6 \\ 6 & -14 & 6 & -43.6 & -4.8 & 0 & -1.2 & 10.4 & -6 & 47.2 \end{bmatrix} \cdot 10^5$$

7. Agregacja - Wektor obciążenia

$$\mathbf{p}_b = \mathbf{B}^{1T} \mathbf{p}_b^1 + \mathbf{B}^{2T} \mathbf{p}_b^2, \quad \mathbf{p} = \mathbf{0}$$

$$\mathbf{p}_b = \begin{bmatrix} 0 \\ -8 \\ 0 \\ -16.5 \\ 0 \\ -7 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} r_7^1 = r_1 \\ r_8^1 = r_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_1^1 = r_7 \\ r_2^1 = r_8 \\ 0 \\ 0 \end{bmatrix}$$

8. Układ równań MES: $\mathbf{Kd} = \mathbf{p} + \mathbf{p}_b$

$$\begin{bmatrix} 16 & -6 & -1.6 & 1.2 & 0 & 0 & -6.4 & -1.2 & -8 & 6 \\ -6 & 28 & -1.2 & 10.4 & 0 & 0 & 1.2 & -24.4 & 6 & -14 \\ -1.6 & -1.2 & 42.4 & -6 & -19.2 & 7.2 & -8 & -6 & -13.6 & 6 \\ 1.2 & 10.4 & -6 & 54.4 & 4.8 & -7.2 & -6 & -14 & 6 & -43.6 \\ 0 & 0 & -19.2 & 4.8 & 19.2 & 0 & 0 & 0 & 0 & -4.8 \\ 0 & 0 & 7.2 & -7.2 & 0 & 7.2 & 0 & 0 & -7.2 & 0 \\ -6.4 & 1.2 & -8 & -6 & 0 & 0 & 16 & 6 & -1.6 & -1.2 \\ -1.2 & -24.4 & -6 & -14 & 0 & 0 & 6 & 28 & 1.2 & 10.4 \\ -8 & 6 & -13.6 & 6 & 0 & -7.2 & -1.6 & 1.2 & 23.2 & -6 \\ 6 & -14 & 6 & -43.6 & -4.8 & 0 & -1.2 & 10.4 & -6 & 47.2 \end{bmatrix} \cdot 10^5 \begin{bmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \\ d_7 \\ d_8 \\ d_9 \\ d_{10} \end{bmatrix} = \begin{bmatrix} 0 \\ -8 \\ 0 \\ -16.5 \\ 0 \\ -7 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} r_1 \\ r_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_7 \\ r_8 \\ 0 \\ 0 \end{bmatrix}$$

9. Uwzględnienie warunków brzegowych

$$\begin{bmatrix} 16 & -6 & -1.6 & 1.2 & 0 & 0 & -6.4 & -1.2 & -8 & 6 \\ -6 & 28 & -1.2 & 10.4 & 0 & 0 & 1.2 & -24.4 & 6 & -14 \\ -1.6 & -1.2 & 42.4 & -6 & -19.2 & 7.2 & -8 & -6 & -13.6 & 6 \\ 1.2 & 10.4 & -6 & 54.4 & 4.8 & -7.2 & -6 & -14 & 6 & -43.6 \\ 0 & 0 & -19.2 & 4.8 & 19.2 & 0 & 0 & 0 & 0 & -4.8 \\ 0 & 0 & 7.2 & -7.2 & 0 & 7.2 & 0 & 0 & -7.2 & 0 \\ -6.4 & 1.2 & -8 & -6 & 0 & 0 & 16 & 6 & -1.6 & -1.2 \\ -1.2 & -24.4 & -6 & -14 & 0 & 0 & 6 & 28 & 1.2 & 10.4 \\ -8 & 6 & -13.6 & 6 & 0 & -7.2 & -1.6 & 1.2 & 23.2 & -6 \\ 6 & -14 & 6 & -43.6 & -4.8 & 0 & -1.2 & 10.4 & -6 & 47.2 \end{bmatrix} \cdot 10^5 \begin{bmatrix} 0 \\ 0 \\ d_3 \\ d_4 \\ d_5 \\ d_6 \\ 0 \\ 0 \\ d_9 \\ d_{10} \end{bmatrix} = \begin{bmatrix} 0 \\ -8 \\ 0 \\ -16.5 \\ 0 \\ -7 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} r_1 \\ r_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ r_7 \\ r_8 \\ 0 \\ 0 \end{bmatrix}$$

Rozwiązanie:

$$\mathbf{d} = \{0 \ 0 \ 3.881 \ -11.03 \ 3.949 \ -19.62 \ 0 \ 0 \ -3.744 \ -10.75\} \cdot 10^{-5} \text{ m}$$

$$\mathbf{r} = \{-54 \ 16.744 \ 0 \ 0 \ 0 \ 0 \ 54 \ 14.756 \ 0 \ 0\} \text{ kN}$$

10. Powrót do elementu

- Element 1

$$\mathbf{d}^1 = \mathbf{B}^1 \mathbf{d} = \{0 \ 0 \ -3.744 \ -10.75 \ 3.881 \ -11.03 \ 0 \ 0\} \cdot 10^{-5}$$

$$\mathbf{e}^1 = \mathbf{B}^1 \mathbf{d}^1$$

$$\mathbf{e}^1 = \begin{bmatrix} 0.953y - 0.936 \\ -0.034x \\ 0.953x - 0.034y - 2.688 \end{bmatrix} \cdot 10^{-5}, \quad \mathbf{e}^1(2, 1) = \begin{bmatrix} 1.708 \\ 6.831 \\ -81.600 \end{bmatrix} \cdot 10^{-7}$$

$$\mathbf{s}^1 = \mathbf{D} \mathbf{e}^1$$

$$\mathbf{s}^1 = \begin{bmatrix} 182.976y - 179.712 - 1.632x \\ 45.744y - 44.928 - 6.528x \\ 68.616x - 2.448y - 193.536 \end{bmatrix}, \quad \mathbf{s}^1(2, 1) = \begin{bmatrix} 0 \\ -12.297 \\ -58.750 \end{bmatrix} \text{ kPa}$$

- Element 2

$$\mathbf{d}^2 = \mathbf{B}^2 \mathbf{d} = \{-3.744 \ -10.75 \ 3.949 \ -19.62 \ 3.881 \ -11.03\} \cdot 10^{-5}$$

$$\mathbf{e}^2 = \mathbf{B}^2 \mathbf{d}^2$$

$$\mathbf{e}^2 = \begin{bmatrix} 3.416 \\ 13.660 \\ -48.610 \end{bmatrix} \cdot 10^{-7}$$

$$\mathbf{s}^2 = \mathbf{D} \mathbf{e}^2$$

$$\mathbf{s}^2 = \begin{bmatrix} 0 \\ -24.593 \\ -35.000 \end{bmatrix} \text{ kPa}$$

11. Wyznaczenie wartości przemieszczeń w wybranych punktach

- Element 1 – punkt $(x^{(1)} = 2, y^{(1)} = 1)$

$$\mathbf{u}^1(x^{(1)}, y^{(1)}) = \mathbf{N}^1(x^{(1)}, y^{(1)}) \mathbf{d}^1$$

$$\mathbf{u}^1(2, 1) = \begin{bmatrix} \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 \\ 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{4} \end{bmatrix} \cdot \begin{bmatrix} 0 \\ 0 \\ -3.744 \\ -10.75 \\ 3.881 \\ -11.03 \\ 0 \\ 0 \end{bmatrix} \cdot 10^{-5} = \begin{bmatrix} 0.0343 \\ -5.4450 \end{bmatrix} \cdot 10^{-5} \text{ [m]}$$

- Element 2 – punkt $(x^{(2)} = 0.5, y^{(2)} = 1.5)$

$$\mathbf{u}^2(x^{(2)}, y^{(2)}) = \mathbf{N}^2(x^{(2)}, y^{(2)}) \mathbf{d}^2$$

$$\mathbf{u}^2(0.5, 1.5) = \begin{bmatrix} \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{2} & 0 \\ 0 & \frac{1}{4} & 0 & \frac{1}{4} & 0 & \frac{1}{2} \end{bmatrix} \cdot \begin{bmatrix} -3.744 \\ -10.75 \\ 3.949 \\ -19.62 \\ 3.881 \\ -11.03 \end{bmatrix} \cdot 10^{-5} = \begin{bmatrix} 1.9916 \\ -13.1065 \end{bmatrix} \cdot 10^{-5} \text{ [m]}$$